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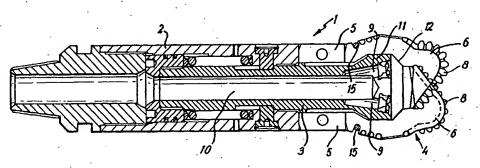
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(54) Title: EXPANDABLE BIT



(57) Abstract: An expandable drill bit for use with earth drilling equipment. The bit includes arms held in a closed position, so that the bit may be inserted through casing or a small bore hole. The arms are expandable to create an expanded drill bit having a crown profile common to a solid crown bit. The arrangement of the arms provides a short gauge length so that the expanded bit is steerable downhole. Embodiments of the expandable drill bit are detailed to show mechanisms for actuating the arms between the open and closed positions.



EXPANDABLE BIT

1 2

3 The present invention relates to a drill bit primarily for use 4 in creating well bores, but which can also be used inside

5 liner casing to remove build-ups of scale.

6

The drilling of wells for oil and gas production is achieved using a string of drill pipe with a drill bit mounted at the lowermost end, which is rotated from the surface into the earth. The drill bit is generally comprised of a body which

11 can be secured to a work string at its uppermost end, ie the

12 shank, and a crown. The crown is essentially the area of the

13 bit which carries the cutting means which drill the earth to

14 create the bore, and is comprised of an uppermost chamfer, an

15 annular gauge and tapered flank upon which the cutting means

16 are mounted, and a lowermost nose which engages with the

17 bottom of the hole.

- 19 After a section of well has been bored, it is common practice
- 20 to insert joints of heavy steel tubing, commonly known as
- 21 casing, into the bore to act as a liner to structurally
- 22 support the walls of the well bore from collapse.

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2 Typically, the casing has a smaller outside diameter than the drill bit which created the bore into which the casing is to be passed.

5 The standard method used to drill well bores is to drill each 6. section with consecutively smaller bits and then line the well 7. bores with proportionately smaller casing. However, a 8 standard practice also exists with regard to using a drilling 9 underreamer positioned behind a standard drill bit acting as a 10 pilot to cut the inner cross-sectional area of the well bore. 11 Conventional underreamers include a number of expandable arms 12 which can move between a withdrawn or closed configuration and 13 an expanded or open configuration. The pilot bit and 14 underreamer can be passed through the casing when the 15 16 underreamer is closed. After passing through the casing the 17 underreamer can be opened in order to enlargen the rat-hole below the casing shoe, and hence create a wellbore equal to or 18 19 larger than the original drilled hole. In recent years bicentre bits; which have offset cutting members mounted at 20 irregular intervals around the crown of the bit, have been 21 22 developed as an alternative to underreamers. However, these 23 bits are unstable due to their irregular structure and tend to 24 be more difficult to control for directional purposes than 25 ordinary drill bits and may not drill the expected swept

28

26

27

by the crown.

29 It will be appreciated that it is not always desirable, or in 30 fact possible to drill a truly straight well bore. 31 example it may be desirable to control the direction of the 32 drilling procedure in order to reach a particular area, or to 33 create a horizontal or expanded well once the correct depth of 34 bore has been drilled. In such instances, it is common to use

diameter of the offset pads which ream the pilot hole created

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steerable drilling apparatus. Standard steerable drilling 1 apparatus is generally comprised of a downhole motor which can 2 drive or rotate a drill bit positioned at the lowermost end of 3 the motor. Typically, the downhole motor has a bent housing 4 with an angle of 0.5 to 2.0 degrees above the bearing section 5 of the motor about 6-10 feet behind the bit. This can be used 6 to steer the assembly when the drill-string is not being 7 rotated and allows the direction of the well-bore to be 8. controlled in response to changing downhole conditions. 9 order to steer the drill bit in a desired direction, rotation 10 of the drill string is stopped which allows the motor to 11 12 incline the drill bit to tilt in the desired direction. As a result, a curved section of the bore can be formed. At other 13 times the drill string is rotated as normal, which negates the 14 action of the downhole motor bent housing on the drill bit. 15 16 17 In general, underreamers and bi-centre bits are not designed 18 for high accuracy open hole directional drilling with 19 steerable downhole motors or rotary steerable systems. Steerable drilling requires the drill bit which is utilised to 20 21 be able to change the direction of the drilled well bore 22 quickly when being tilted or a side force is applied. 23 Underreamers have a large spacing between the pilot bit and 24 the expandable arms and therefore do not permit this rapid 25 directional change to take place. Bi-centre bits are designed 26 such that the distance between the crown and offset pads is relatively large, and as a consequence these bits are not as 27 28 steerable as ordinary short gauge bits. 29 30

It is recognised in the present invention that it would be an 31 advantage to provide a truly expandable drill bit which is 32 small enough such that it can be passed through a small 33 diameter bore or casing in one mode and then can be expanded 34 such that it can drill a larger diameter hole below the

restriction it has passed through in a second mode, but wherein the drill bit is designed such that it has a 2 sufficiently short gauge length to be used in a variety of drilling operations including steerable drilling applications. 6 It is therefore an object of the present invention to provide . 7. a truly expandable drill bit which can be used with steerable downhole motors or rotary steerable systems. 8 It is a further object of the present invention to provide an 10 expandable drill bit which, when expanded, has a short gauge 11 12 length and a crown profile with a shape common to solid crown 13 bit, and therefore has the same steerability as conventional 14 steerable solid crown drill bits. 15 16 According to the present invention there is provided an 17 expandable drill bit for use with earth drilling equipment, 18 wherein the drill bit is comprised of a body having two or 19 more arms, the arms being provided by the crown of the drill 20 bit having a split crown profile, wherein the arms support a 21 plurality of cutting elements and are hingeably attached to 22 the body, and wherein the arms are moveable between a first 23 and second position, wherein the arms are closed in the first 24 position and expanded in the second position. 25 26. Preferably, when the arms are in the second expanded position, 27 the drill bit has a short gauge length and the profile of the 28 expanded crown is similar to that of a steerable solid crown 29 bit.

30.

Preferably, movement of the arms from the first closed 31 32 position to the second expanded position is provided by virtue 33 of the movement of an actuating shaft.

1 In one preferred embodiment movement of the actuating shaft in

- 2 a downward direction drives the arms from the first closed
- 3 position to the second expanded position.

4

- 5 In an alternative embodiment, movement of the actuating shaft
- 6 in an upward direction lifts the arms from the first closed
- 7 position to the second expanded position.

8.

- 9 The first embodiment is preferred as the actuating shaft can
- 10 be used to support the arms to a greater degree. Also the bit
- 11 nozzles can be placed closer to the cutters for greater
- 12 hydraulic effect and the thrust area and hence the axial
- 13 thrust acting on the actuating shaft to push the arms open can
- 14 be made much greater, while minimising the overall length of
- 15 the bit for greater steerability.

16

- 17 Preferably movement of the actuating member is driven by a
- 18 hydrodynamic pressure drop.

19

- 20 Most preferably said hydrodynamic pressure drop is created by
- 21 one or more nozzles which may be attached to the lowermost end
- 22 of the actuating member.

23

- 24 Preferably the one or more nozzles communicate with a through
- 25 bore defined by the actuating member.

26

- 27 Preferably the actuating member has an external upset at its
- 28 lowermost end which supports the arms when closed in the first
- 29 position, and drives the arms to the second expanded position
- 30 upon the application of hydraulic pressure created by
- 31 directing mud flowing through the ports or nozzles in the
- 32 actuating member.

33.

6

- 1 Preferably the arms have an internal profile which
- 2 communicates with the upset end of the actuating member such
- 3 that the upset end of the actuating member supports the arms
- 4 both in the first closed position and in the second expanded
- 5 position.

6

- 7 Preferably the drill bit is adapted for use with steerable
- 8 drilling apparatus. The steerable drilling apparatus may
- 9 include a downhole motor.

10

- 11 In one embodiment the arms are driven from the second expanded
- 12 position to the first closed position by the action of return
- 13 springs.

14

- 15 Optionally a first return spring is a heavy duty helical coil
- 16 spring.

17

- 18 Alternatively a stack of disc springs can be utilised as the
- 19 first return spring.

20

- 21 Preferably a second return spring comprises a single coil
- 22 split ring.

23

- 24 Preferably the second return spring is located externally to
- 25 the arms.

26

- 27 In a second embodiment the arms are pulled together from the
- 28 second expanded position to the first closed position with the
- 29 aid of both secondary return springs, wherein the springs are
- 30 located internally to the arms.

31

- 32 Preferably the cutting elements comprise one or more rows of
- 33 cutters on each arm.

7

1 Typically the cutters are made from a hard material such as

2 diamond or tungsten carbide.

.3

4 Preferably the cutters are arranged to form a double row of

- 5 cutters in the centre of the bit, ie at least two of the arms
- 6 overlap when in the closed position and when in the second
- 7 expanded position the cutters will cut the full swept area out
- 8 to the expanded gauge diameter.

9

- 10 Optionally the arms may include a sensor to detect if the arms
- 11 are out to the gauge diameter intended. The sensor activation
- 12 can also confirm that the arm is still in place, ie has not
- 13 been torn off.

14

- 15 Preferably said sensor is in the form of an electrical switch
- 16 to complete a circuit and one would preferably be used for
- 17 each arm.

18

- 19 Optionally the drill bit may contain a sensor which registers
- 20 the travel of the actuating shaft or the actuating shaft
- 21 coupling.

22

- 23 Embodiments of the present invention will now be illustrated,
- 24 by way of example, with reference to the following Figures in
- 25 which:

- 27 Figure 1 illustrates an expandable drill bit in a closed
- 28 configuration and in cross sectional detail in accordance with
- 29 a first embodiment of the present invention;
- 30 Figure 2 illustrates the expandable drill bit of Figure 1 in
- 31 an expanded configuration in cross sectional detail;
- 32 Figure 3 illustrates the crown of the expandable drill bit in
- 33 cross section, perpendicular to the view in Figure 1;

1 Figure 4 illustrates the crown of the expandable drill bit in

- 2 cross section, perpendicular to the view in Figure 2;
- 3 Figure 5 illustrates an elevated view of the crown of the
- 4 expandable drill bit in a closed configuration;
- 5 Figure 6 is an elevated view of the crown of the expandable
- 6 drill bit in an expanded configuration;
- 7 Figures 7 and 8 illustrate the hinge upon which the arms of
- 8 the expandable drill bit are mounted;
- 9 Figure 9 illustrates an expandable drill bit in a closed
- 10 configuration and in cross-sectional detail in accordance with
- 11 a second embodiment of the present invention; and
- 12 Figure 10 illustrates the expandable drill bit of Figure 9 in
- 13 an expanded configuration and in cross-sectional detail.

- 15 Referring firstly to Figure 1, an expandable drill bit is
- 16 depicted at 1 and is comprised of a generally cylindrical body
- 17 2, which can be attached to a work string (not shown) by
- 18 either a pin or box threaded end connection, and an actuating
- 19 member 3 shown as a shaft. The drill bit 1 also comprises four
- 20 arms which are arranged as pairs and are formed as a result of
- 21 the lowermost end of the drill bit 1 having a split crown
- 22 profile 4, which can be seen from the elevated view of the bit
- 23 1 in Figures 5 and 6. More specifically, the split crown 4
- 24 comprises two pairs of segments or arms, each arm of which is
- 25 attached to a hinge 5 which allows the arm to swing out from
- 26 the body 2 of the bit 1. An individual hinge 5 with a pin
- 27 inserted, can be seen in more detail in Figure 7 upon which an
- 28 individual arm of the drill bit 1 rests upon. In the cross
- 29 sectional depiction of the drill bit in Figure 1, one pair of
- 30 arms 6 can be seen. A second pair of arms 7, as seen in
- 31 Figures 3 and 4 extend perpendicularly to the pair of arms 6
- 32 shown in Figure 1. The arms 6 and 7 are fitted with a
- 33 plurality of cutting elements 8 made of a hard material,
- 34 typically tungsten carbide or polycrystalline diamond which

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1 contact and drill the earth when the arms 6 and 7 are in an

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- 2 expanded configuration. However the arms 6 and 7 have an
- 3 external profile such that when they are collapsed or closed
- 4 into the body 2 of the bit 1, the cutting elements 8 do not
- 5 ream the casing bore. Each arm 6 and 7 may carry a single
- 6 or double row of cutters. The arms 6 and 7 may also be
- 7 designed such that in the closed position shown in Figure 5,
- 8 there is a double row of cutters mounted back to back in the
- 9 centre of the bit 1 to protect and supply a cutting action for
- 10 drilling when the arms 6 and 7 are in a closed position. The
- 11 arms 6 and 7 form a T shape around the hinge pin 5 area, which
- 12 prevents them from being left downhole if the hinge pin 5
- 13 breaks.

14

- 15 Nozzles 9 are attached to the lowermost end of the actuating
- 16 shaft 3 and communicate with a fluid through bore 10 which is
- 17 defined by the body of the actuating shaft 3. The nozzles 9
- 18 may be permanently or detachably fixed to the actuating shaft
- 19 3 to allow the jetting of drilling fluid. In the depicted
- 20 embodiment a total of four nozzles 9 are fitted to the
- 21 actuating shaft 3 although it is recognised that the number of
- 22 nozzles 9 which can be fitted is not limited, and is
- 23 restricted only by the space constraints of the size of the
- 24 actuating shaft 3. The nozzles 9 are used for standard
- 25 jetting of the bit face when drilling, to remove any cutting
- 26 build up which may gather immediately in front of the
- 27 actuating member 3 and arms 6 and 7, and also to supply a
- 28 hydraulic pressure drop which moves the actuating shaft 3.

29.

- 30 The arms 6 and 7 of the drill bit 1 can move between a first
- 31 position shown in Figure 1 wherein they are closed against the
- 32 body 2 of the drill bit 1, and a second position shown in
- 33 Figure 2 wherein they are expanded away from the body 2.
- 34 Movement of the arms 6 and 7 from the first closed to the

10

second expanded position occurs when a pressure drop is created across the assembly of nozzles 9, thereby moving the 3 actuating shaft 3 downwards. The actuating shaft 3 drives the arms 6 and 7 outwards to their gauge stop position and acts to 5 support and reinforce the arms 6 and 7 and hinge pins 5. will be seen from Figures 1 to 4 that the lowermost end of the -6 actuating member 3, that is the end nearest to the crown of : 7. the drill bit 1, has an external upset 11. The arms 6 and 7 have a corresponding internal profile 12 which communicates 9 with the upset end 11 of the actuating member 3 (Figures 3 and 10 11 When it is desirable for the arms 6 and 7 to be expanded 12 (Figure 4), fluid is passed into the actuating shaft 3 through 13 bore 10 and through the nozzles 9 creating a sufficient 14 hydrodynamic pressure drop to move the actuating shaft 3 in a downward direction. As a result the upset end 11 of the 15 actuating member 3 will move down in the direction of the arrow shown in Figure 3 to communicate with a corresponding 17 18 shoulder located in the internal profile 12 of the arms 4 as 19 seen in Figure 4, thereby driving the arms 6 and 7 outwards 20 into the second expanded position. The actuating member 3 21 supports the arms 6 and 7 when expanded, from the inward force 22 which is impacted on them by the walls of the bore. In order 23 to retain the arms 6 and 7 in the closed position, the flow 24 rate through the nozzles 9 is minimised in order to keep the 25 hydrodynamic pressure below that which is required to drive 26 the actuating shaft 3 in a downwards direction to expand the 27 arms 6 and 7. A shear pin may also be incorporated into the 28 bit 1 between each arms 6 and 7 and the actuating shaft 3 or 29 between the actuating shaft 3 and the body 2. 30 31

In the described embodiment the hydrodynamic pressure drop 32 causes the actuating member 3 to move in a downward direction 33 where it engages with an internal profile shoulder 12 on the 34 arms 6 and 7 to expand them outwardly from the body 2 of the

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11. drill bit 1. However it is recognised that in an alternative 1 embodiment of the present invention the actuating shaft 3 may 2 be adapted to be driven in an upward direction by the pressure 3 drop, wherein upon moving upwards, the actuating member 3 4 lifts the arms 6 and 7 into an expanded open configuration. 5 6 The actuating shaft 3 is prevented from rotating with respect 7 to the body 2 by four (by way of example) pins so that the nose of the actuating shaft will strengthen the four arms when torque is applied to them. A spline could also be used. The 10 nose of the actuating shaft 3 has a milled profile to support 11 12 the arms with respect to torque applied when drilling. 13 The back of the arms 6 and 7 is designed such that it has a 14 15 low angle with respect to the hole diameter. This allows 16 maximum force to be applied in the event that the arms 6 and 7 17 stick in the second expanded position so that when the drill 18 bit 1 is pulled up against the casing shoe(not shown) the arms 19 6 and 7 will be driven back against the body 2 of the drill 20 bit 1 with maximum force. This tapered surface could also 21 have cutters fitted for back-reaming when pulling out of hole. 23 It will be appreciated that at some point prior to running the apparatus it may be necessary to check the size of the nozzles 9 in order to determine whether they suit the required downhole hydraulics for the run. In the preferred embodiment the drill bit 1 will be nozzled such that the arms 6 and 7

22

34

24 25

26

27

28 begin to extend at a minimum hydrodynamic pressure of

29 approximately 100psi and be fully expanded by 200 psi,

30 although it will be appreciated that these pressures could be

31 varied for the particular drilling application and conditions.

32 This allows a minimum circulation to be run through the bit 1

33 for lubrication, without expanding the arms 6 and 7.

12

In order to change the nozzles 9 prior to use, a threaded rod 13 already screwed into a coupling is inserted into the fluid 2 3 through bore 10 of the drill bit 1, as can be seen in Figure 2. The coupling is screwed onto the drill bit 1, typically 4 onto the inlet pin or box thread which connects the drill bit 1 to a work string (not shown) in use. The actuating shaft 3 .6 can then be driven downwards by rotating the threaded rod 13 . 7 into the coupling in order to drive the arms 6 and 7 away from the body 2, permitting access to the nozzles 9 which are located between the arms 6 and 7 on the expanded face of the 10 11 bit 1 (Figure 6). The nozzles 9 can be removed and replaced using a standard bit nozzle spanner (not shown). 13 In order to allow the drill bit 1 to pass through 14 restrictions, such as a narrow diameter bore or in-place casing, it is necessary for the arms 6 and 7 of the drill bit 17 1 to be closed. This is achieved by way of two springs which 18 drive the arms 6 and 7 back into the body 2. The first spring

19 14 is an internal heavy duty helical coil spring whilst the

20 second is a single coil split ring 15 which is mounted around

21 the outside of the four arms 6 and 7, in the area just outside

22 the hinge pins 5. The second spring 15 adds a more positive

23 return force directly to the arms 6 and 7 when the actuating

24 member 3 returns to the position shown in Figure 3.

25

32

26 It is recognised that although the springs are located 27 external to the arms 6 and 7 in the described embodiment, in

28 an alternative embodiment two or more springs could be used on

29 the inside of the arms 6 and 7 which pull them together.

30 Further, the first spring could alternatively be a stack of

31 disc springs. PCT/GB01/01814

13.

1 Figure 8 illustrates a sectional view through the hinge

- 2 section of the drill bit 1. In Figure 8 four hinges 5, can be
- 3 seen in position around the actuating shaft 3.

4.

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- 5 The hinges 5 are positioned between the body 2 of the tool and
- 6 the arms (not shown), each arm being attached to a hinge 5
- 7 which allows the arm to expand away from the body 2 upon
- 8 movement of the actuating shaft 3.

9

- 10 Note also that each pair of arms could be linked via a guide
- 11 pin with one of the arms having a pin rigidly fitted with a
- 12 slot in the adjacent mating arm.

13

- 14 The drill bit 1 also preferably comprises low friction piston
- 15 seals which may be PTFE seals with O ring energisers, between
- 16 the body 2 and the shaft 3, which minimise the force available
- 17 from the coil spring 15 to return the actuating shaft 3. In a
- 18 preferred embodiment the bore 10 of the body 2 has a corrosion
- 19 resistant coating or treatment so that the seals run on a
- 20 smooth surface.

21

- 22 Figures 9 and 10 illustrate an expandable drill bit according
- 23 to a second embodiment of the present invention. Like parts
- 24 to those of the first embodiment shown in Figures 1 through 8
- 25 are given the same reference numerals, but are suffixed "A".

26

- 27 Expandable bit 1A is now such that the drilling load applied
- 28 to the bit is taken entirely through the inner
- 29 mandrel/actuating shaft 3A. This means that the application
- 30 of drilling weight to the bit now keeps the arms 6A, 7A in the
- 31 expanded position in addition to the hydraulic force acting on
- 32 an internal piston 18.

The tool 1A is hydraulically actuated due to the pressure drop created by throttling the flow of drilling fluid by the nozzles 9A in the head 16 of the bit. Simply applying drilling weight to the tool 1A in the closed position would also tend to expand the arms 6A, 7A, but is not a principal operating feature. Internal hydraulic pressure is applied to the chamber 17 above 8 the piston 18 mounted on the inner mandrel 3A by means of radial drilled holes 19 in the mandrel 3A. The force created 10 moves the outer cylinder 2A axially upwards, compressing the 11 spring 14A and drawing the arms 6A, 7A upwards over the profile of the head 16 into the expanded position. 13 1.4 The arms 6A, 7A are now constrained within slots 20 in the head for greater rigidity. Guide pins 21 act on slots 20 17 machined in the arms 6A, 7A to ensure that the arms 6A, 7A 18 return to the closed position on removal of the pressure 19 differential, as described hereinbefore. Note that a 20 secondary spring is no longer used to close the arms 6A, 7A. 21 An additional feature of the second embodiment of bit 1A is 22 23 that pulling upwards on the tool 1A will tend to drag the 24 external sleeve 2A downwards, thus moving the arms 6A, 7A to 25 the closed position. 26 27 A further feature of the second embodiment of bit 1A includes 28

two sensors 22, 23.

29

30 Arms 6A and 7A are fitted with sensors 22A-D. Sensors 22A-D 31 are electronic sensors, which signal when the arms 6A and 7A 32 are out at gauge size. This signal is sent back into an MWD 33 tool behind the bit 1A or may be an instrumented downhole motor, and then transmitted directly to the surface, so that

1 the operator is aware of the configuration of the bit 1A as it. is run downhole. The sensors 22A-D being activated would also confirm the arms 6A and 7A are still in position ie have not 3 been torn off. Sensor 23 is also fitted to bit 1A. registers the movement of the actuating shaft 3A. The advantage of the present invention over the prior art is that there is provided a truly expandable drill bit, in contrast to an offset bi-centre bit or an underreamer for use in conjunction with a standard drill bit. The expandable 10 drill bit is therefore characterised in that it has all the 11 proven characteristics of a standard steerable drill bit, most 12 notably a short gauge length with a standard crown profile 13 14 shape and can be used with steerable drilling apparatus, but also has a variable diameter which facilitates the passage of 15 the drill bit through an area of a well bore or casing with a 16 17 restricted diameter in order to drill a section of bore with a 18 greater diameter, below the restricted area.

19

20 Further modifications and improvements may be incorporated 21 without departing from the scope of the invention herein 22 intended.

CLAIMS

1 2

An expandable drill bit for use with earth drilling 3 equipment, wherein the drill bit is comprised of a body 4 having two or more arms, the arms being provided by the 5 crown of the drill bit having a split crown profile, . 6 wherein the arms support a plurality of cutting elements 7. and are hingeably attached to the body, and wherein the 8 arms are moveable between a first and second position, 9 wherein the arms are closed in the first position and 10 11 expanded in the second position.

12

2. An expandable drill bit as claimed in Claim 1, wherein
when the arms are in the second expanded position, the
drill bit has a short gauge length and the profile of the
expanded crown is similar to that of a steerable solid
crown bit.

18

20 An expandable drill bit as claimed in Claim 1 or Claim 2, wherein movement of the arms from the first closed position to the second expanded position is provided by virtue of the movement of an actuating shaft.

23

24 4. An expandable drill bit as claimed in Claim 3, wherein
25 movement of the actuating shaft in a downward direction
26 drives the arms from the first closed position to the
27 second expanded position.

28

29 5. An expandable drill bit as claimed in Claim 3, wherein
30 movement of the actuating shaft in an upward direction
31 lifts the arms from the first closed position to the
32 second expanded position.

1 6. An expandable drill bit as claimed in any one of Claims 3.
2 to 5, wherein movement of the actuating shaft is driven
3 by a hydrodynamic pressure drop.
4
5 7. An expandable drill bit as claimed in Claim 6, wherein
6 said hydrodynamic pressure drop is created by one or more
7 nozzles which are attached to a lowermost end of the
8 actuating shaft.

9

10 8. An expandable drill bit as claimed in Claim 7, wherein
11 the one or more nozzles communicate with a through bore
12 defined by the actuating shaft.

13

9. An expandable drill bit as claimed in Claim 7 or Claim 8, wherein the actuating member has an external upset at the lowermost end which supports the arms when closed in the first position, and drives the arms to the second expanded position upon the application of hydraulic pressure created by directing mud flowing through the ports or nozzles in the actuating shaft.

21

22 10. An expandable drill bit as claimed in Claim 9, wherein
23 the arms have an internal profile which communicates with
24 the upset end of the actuating shaft such that the upset
25 end of the actuating shaft supports the arms both in the
26 first closed position and in the second expanded
27 position.

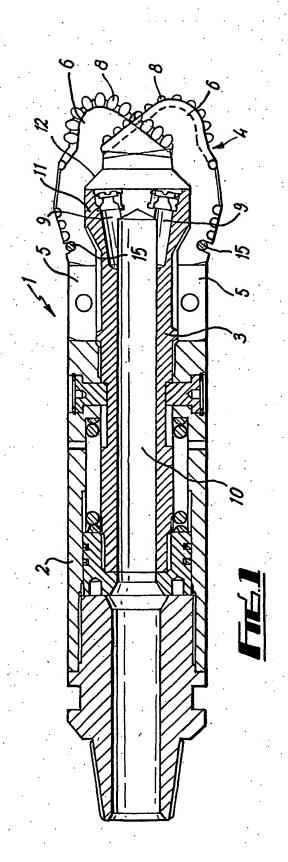
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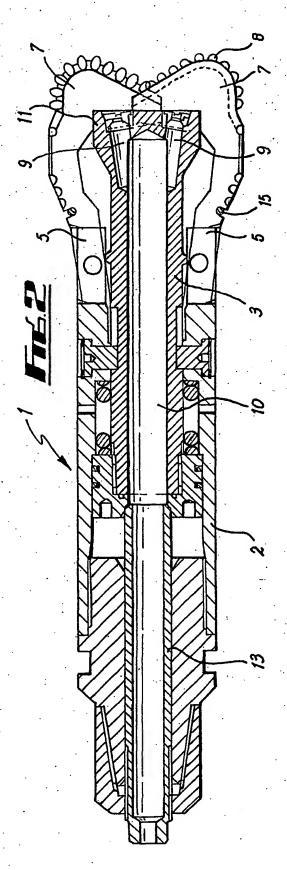
29 11. An expandable drill bit as claimed in any preceding 30 Claim, wherein the drill bit is adapted for use with 31 steerable drilling apparatus.

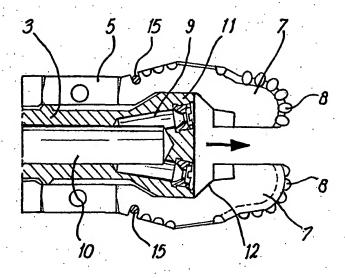
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33 12. An expandable drill bit as claimed in any preceding Claim, wherein the arms are driven from the second

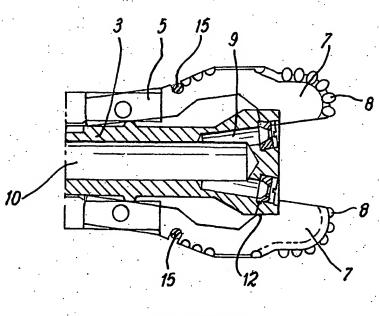
1	··	expanded position to the first closed position by the
2	•	action of return springs.
3		
4	13.	An expandable drill bit as claimed in Claim 12, wherein
5		first return spring is a heavy duty helical coil spring.
6		
7	14.	An expandable drill bit as claimed in Claim 12 or Claim
8		13, wherein a second return spring comprises a single
٠9		coil split ring.
10		
11	15.	An expandable drill bit as claimed in Claim 14, wherein
12		the second return spring is located externally to the
13		arms.
14		
15	16.	An expandable drill bit as claimed in any preceding
16	٠.	Claim, wherein the cutting elements comprise one or more
1 7		rows of cutters on each arm.
18	•	
19	17.	An expandable drill bit as claimed in Claim 16, wherein
20		the cutters are arranged to form a double row of cutters
21		in the centre of the bit.
22		
23	18.	An expandable drill bit as claimed in any preceding
24		Claim, wherein the arms include a sensor to detect if th
2,5	•	arms are out to the gauge diameter intended.
26		
27	19.	An expandable drill bit as claimed in Claim 18, wherein
28		said sensor is in the form of at least one electrical
29		switch to complete a circuit in each arm.
30		
31	20.	An expandable drill bit as claimed in Claim 20, wherein
32		the drill bit contains a sensor which registers the
33		travel of the actuating shaft.



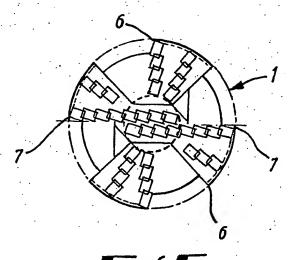


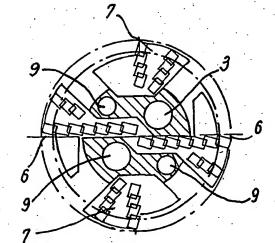


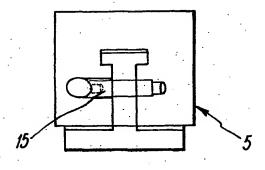




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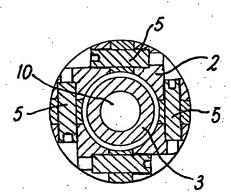




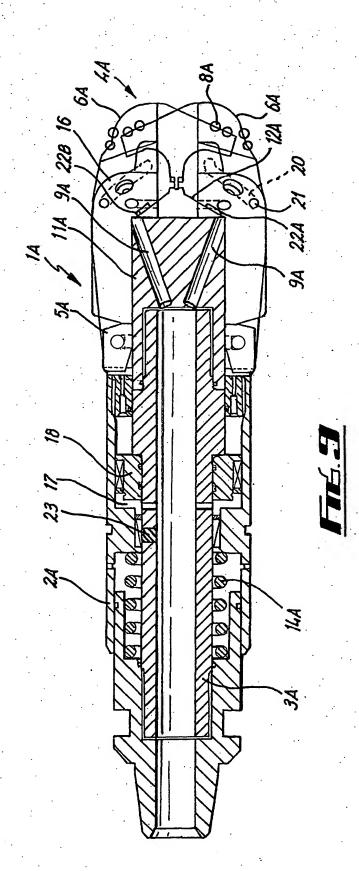


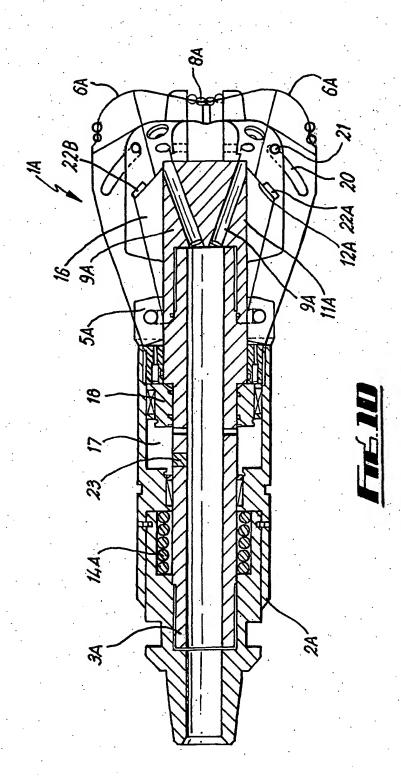






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INTERNATIONAL SEARCH REPORT

ational Application No rt[/GB 01/01814

A. CLASSIFICATION OF SUBJECT MATTER
IPC 7 E21B10/32 E21B10/62

According to International Patent Classification (IPC) or to both national classification and IPC

Minimum documentation searched (classification system followed by classification symbols) IPC $\,7\,$ E21B

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

EPO-Internal, WPI Data, PAJ, TULSA

Category °	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	US 5 141 063 A (QUESENBURY JIMMY B) 25 August 1992 (1992-08-25) column 2, line 22 -column 3, line 25; figures 3-5	1,3,4,6, 16
Υ		5,7-10, 12,13
X	GB 2 275 067 A (BAKER HUGHES INC) 17 August 1994 (1994-08-17) page 3, line 3 - line 18; figures 1,3,17 page 5, line 12 - line 20 page 20, line 6 - line 24	1,2,11,
Y	US 4 081 042 A (JOHNSON GARY R ET AL) 28 March 1978 (1978-03-28) column 4, line 59 -column 5, line 54; figures 1,2 column 6, line 41 - line 61	5,9,10

Further documents are listed in the continuation of box C.	γ Patent family members are listed in annex.
Special categories of cited documents: A document defining the general state of the art which is not considered to be of particular relevance E earlier document but published on or after the international filing date L document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified) O document referring to an oral disclosure, use, exhibition or other means P document published prior to the International filing date but later than the priority date claimed	 *T* later document published after the International filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention. *X* document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone. *Y* document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art. *&* document member of the same patent family
Date of the actual completion of the international search	Date of mailing of the international search report
6 August 2001	13/08/2001
Name and mailing address of the ISA European Palent Office, P.B. 5818 Palentiaan 2 NL – 2280 HV Rijswijk Tel. (+31-70) 340-2040, Tx. 31 651 epo nl, Fax: (+31-70) 340-3016	Authorized officer Dantinne, P

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	claim 1; figure 4	
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